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# MELTING HEAT TRANSFER IN A NANOFLUID FLOW PAST A PERMEABLE CONTINUOUS MOVING SURFACE

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## Abstract:

A boundary layer analysis is presented for the warm, laminar nanoliquid flow to a melting surface moving parallel to a uniform free stream. The resulting system of non-linear ordinary differential equations is solved numerically using Runge-Kutta method with shooting techniques. Numerical results are obtained for the velocity, temperature and concentration distributions, as well as the friction factor, local Nusselt number and local Sherwood number for several values of the parameters, namely the velocity ratio parameter, melting parameter and nanofluid parameters. The obtained results are presented graphically and in tabular form and the physical aspects of the problem are discussed.

Keywords: Suction/injection, moving surface, nanofluid, boundary layer, shooting technique.

# NOMENCLATURE

С	specific heat	Greek Symbols:	
$D_B$	Brownian diffusion coefficient	$\alpha_{m}$	thermal diffusivity of porous medium
$D_{T}$	thermophoretic diffusion coefficent	η	dimensionless distance
f	reduced stream function	θ	dimensionless temperature
g	gravitational acceleration	μ	viscosity of fluid
$\mathbf{k}_{\mathrm{m}}$	effective thermal conductivity of the	$\rho_{\rm f}$	fluid density
	porous material	$\rho_p$	nano-particle mass density
K	permeability of porous medium	$(\rho C)_{\rm f}$	heat capacity of the fluid
Le	Lewis number	$(\rho C)_m$	effective heat capacity of porous medium
М	melting parameter	$(\rho C)_p$	effective heat capacity of nano-particle
N <sub>b</sub>	Brownian motion parameters		material
Nt	thermophoresis parameters	τ	ratio between the effective heat capacity
Nu	Nusselt number		the fluid
р	pressure	с	nano-particle volume fraction
q	wall heat flux	C	nano-particle volume fraction at the wall
Re	Reynolds number	C <sub>W</sub>	of the plate
Т	temperature	$\mathbf{c}_{\infty}$	ambient nano-particle volume fraction
T <sub>m</sub>	melting surface temperature	Ψ	stream function
T <sub>0</sub>	solid surface temperature		
$T_{\infty}$	ambient temperature	Subscripts	
$\mathbf{u}_{\infty}$	freestream velocity	В	Blasius problem
u, v	Darcy velocity components	S	Sakiadis problem
(x, y)	Cartesian coordinates	W	Refers to condition at wall
		$\infty$	Refers to condition far from the wall

# 1. Introduction

The study of convective heat transfer in nanofluids is gaining a lot of attention. The nanofluids have many applications in the industries since materials of nanometer size have unique physical and chemical properties. Nanofluids are solid-liquid composite materials consisting of solid nanoparticles or nanofibers with sizes typically of 1-100 nm suspended in liquid. Nanofluids have attracted great interest recently because of reports of greatly enhanced thermal properties. For example, a small amount (< 1% volume fraction) of Cu nanoparticles or carbon nanotubes dispersed in ethylene glycol or oil is reported to increase the inherently poor thermal conductivity of the liquid by 40% and 150% respectively by Eastman et al. (2001) and Choi et al. (2001). Conventional particle-liquid suspensions require high concentration (>10%) of particles to achieve such enhancement. However, problems of rheology and stability are amplified at high concentration, precluding the widespread use of conventional slurries as heat transfer fluids. In some cases, the observed enhancement in thermal conductivity of nanofluids is orders of magnitude larger than predicted by well-established theories. Other perplexing results in this rapidly evolving field include a surprisingly strong temperature dependence of the thermal conductivity as reported by Patel et al. (2003) and a three-fold higher critical heat flux compared with the base fluids as reported by You et al. (2003). These enhanced thermal properties are not merely of academic interest. If confirmed and found consistent, they would make nanofluids promising for application in thermal management. Furthermore, suspensions of metal nanoparticles are also being developed for other purposes, such as medical applications including cancer therapy. The interdisciplinary nature of nanofluid research presents a great opportunity for exploration and discovery at the frontiers of nanotechnology.

The characteristics of flow and heat transfer of a viscous and incompressible fluid over flexed or continuously moving flat surfaces in a moving or a quiescent fluid are well understood. These flows occur in many manufacturing processes in modern industry, such as hot rolling, hot extrusion, wire drawing and continuous casting. For example, in many metallurgical processes such as drawing of continuous filaments through quiescent fluids and annealing and tinning of copper wires, the properties of the end product depends greatly on the rare of cooling involved in these processes. Sakiadis (1961) was the first one to analyze the boundary layer flow on continuous surfaces. Crane (1972) obtained an exact solution the boundary layer flow of Newtonian fluid caused by the stretching of an elastic sheet moving in its own plane linearly. Tsou et al. (1967) extended the research to the heat transfer phenomenon of the boundary layer flow on a continuous moving surface. Schowalter (1960) applied the boundary layer theory into power law pseudoplastic fluids and developed two and three dimensional boundary layer equations of the momentum transfer. Acrivos (1960) analyzed the momentum and heat transfer of non-Newtonian fluid past arbitrary external surfaces. Howell et al. (1997) and Rao et al. (1999) investigated momentum and heat transfer phenomena on a continuous moving surface in power law fluids, Magyari and Keller (1999) have studied the thermal boundary layer of moving surfaces. Wang (1989) studied free convection from a vertical stretching surface. Gorla and Sidawi (1994) studied the characteristics of flow and heat transfer from a continuous surface with suction and blowing.

A survey of convective heat transfer in nanofluids has been presented by Buongiorno and Hu (2005) as well as Kakac and Pramuanjaroenkij (2009). Numerous models have been proposed to study convective flows of nanofluids reported by Tiwari et al. (2007), Wang and Wei (2009), Nield and Kuznetsov (2009) and Gorla et al. (2011).

Phase change heat transfer finds applications in magma solidification, permafrost melting, preparation of semiconductor materials etc. The analogy between melting and diffusion mass transfer or transpiration cooling was noted by Huang and Shih (1975). Pedroso and Domoto (1973) developed a method for calculating melting rates based on the diffusion/melting analogy. We present here a similarity analysis for the problem of steady boundary-layer flow and heat transfer from a warm, laminar liquid flow to a melting surface moving parallel to a constant free stream of a nanofluid. It is assumed that the melting of the plate takes place at a steady state. The development of the velocity, temperature and concentration distributions have been illustrated for several values of nanofluid parameters, Prandtl number, Lewis number, velocity ratio and suction/injection parameters.

#### 2. Analysis

Consider a flat surface melting at a steady rate into a constant property warm liquid of the same material. The surface is assumed to be moving at a constant velocity  $u_w$  in a parallel direction to a free stream of a nanofluid of uniform velocity  $u_{\infty}$ . The flow model and coordinate system are shown in Figure 1. Either the surface velocity or the free-stream velocity may be zero but not both at the same time. The physical properties of the fluid are

assumed to be constant. It is assumed that the temperature of the melting surface is  $T_m$  while the free stream temperature  $T_{\infty} > T_m$ . Under such condition, the governing equations of the steady, laminar boundary-layer flow on the moving surface are given by:



Figure 1: Flow Model and Coordinate System

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0 \tag{1}$$

$$u\frac{\partial u}{\partial x} + v\frac{\partial u}{\partial y} = -\frac{1}{\rho_{\rm f}}\frac{\partial p}{\partial x} + v\left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2}\right)$$
(2)

$$u\frac{\partial v}{\partial x} + v\frac{\partial v}{\partial y} = -\frac{1}{\rho_{\rm f}}\frac{\partial p}{\partial y} + v\left(\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2}\right) \tag{3}$$

$$u\frac{\partial T}{\partial x} + v\frac{\partial T}{\partial y} = \alpha \left(\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2}\right) + \tau \left\{ D_B \left(\frac{\partial c}{\partial x}\frac{\partial T}{\partial x} + \frac{\partial c}{\partial y}\frac{\partial T}{\partial y}\right) + \left(\frac{D_T}{T_{\infty}}\right) \left[ \left(\frac{\partial T}{\partial x}\right)^2 + \left(\frac{\partial T}{\partial y}\right)^2 \right] \right\}$$
(4)

$$u\frac{\partial c}{\partial x} + v\frac{\partial c}{\partial y} = D_{B}\left(\frac{\partial^{2} c}{\partial x^{2}} + \frac{\partial^{2} T}{\partial y^{2}}\right) + \left(\frac{D_{T}}{T_{\infty}}\right)\left(\frac{\partial^{2} T}{\partial x^{2}} + \frac{\partial^{2} T}{\partial y^{2}}\right)$$
(5)

The boundary conditions are given by,

$$y = 0: \ u = u_{w}, v = v_{w}(x), \ k \frac{\partial T}{\partial y} = \rho_{f} [\lambda + C_{s} (T_{m} - T_{0})] v_{w}, c = c_{w}$$

$$y \to \infty: \ u = u_{\infty}, T = T_{\infty}, c = c_{\infty}$$
(6)
The boundary condition of  $u = u_{w}$  in (6) represents the case of a plane surface moving in parallel to the free

stream. To analyze the effect of both the moving and the free stream on the boundary-layer flow, we propose a new similarly coordinate and a dimensionless stream function

$$\eta = \frac{y}{x} (Re_w + Re_\infty)^{\frac{1}{2}}, \qquad f = \frac{\psi}{\frac{v(Re_w + Re_\infty)^{\frac{1}{2}}}{(Re_w + Re_\infty)^{\frac{1}{2}}}}$$
(7)

which are the combinations of the traditional ones:

$$\eta_B = \frac{y}{x} R e_{\infty}^{\frac{1}{2}}, \qquad f_B = \frac{\psi}{v R e_{\infty}^{\frac{1}{2}}}$$
(8)

for the Blasius problem (stationary wall and uniform freestream velocity) and

$$\eta_{S} = \frac{y}{x} R e_{w}^{\frac{1}{2}}, \qquad f_{S} = \frac{\psi}{v R e_{w}^{\frac{1}{2}}}$$
(9)

from the Sakiadis (uniformly moving wall with stagnant freestream) problem.

The Reynolds numbers are defined as:

$$Re_{w} = \frac{u_{w}x}{v}, \qquad Re_{\infty} = \frac{u_{\infty}x}{v}$$
(10)

$$\gamma = \frac{u_{w}}{(u_{w}+u_{\infty})} = \left(1 + \frac{u_{\infty}}{u_{w}}\right)^{-1} = \left(1 + \frac{Re_{\infty}}{Re_{w}}\right)^{-1} , \qquad (11)$$

Note that form the Blasius problem,  $u_w = 0$  therefore  $\gamma = 0$ . On the other hand, for the Sakiadis problem,  $u_{\infty} = 0$  and thus  $\gamma = 1$ . In addition, we also define dimensionless temperature and concentration function as

$$\theta = \frac{T - T_{\infty}}{T_m - T_{\infty}} \tag{12}$$

$$\phi = \frac{1}{c_w - c_\infty} \tag{13}$$

Using the transformation variables defined in Equations (7) - (13), the governing transformed equations may be written as

$$f_{\mu}^{\prime\prime\prime} + \frac{ff^{\prime\prime}}{2} = 0 \tag{14}$$

$$\frac{\theta^{\prime\prime}}{Pr} + \frac{f\theta^{\prime}}{2} + N_b \phi^{\prime} \theta^{\prime} + N_t (\theta^{\prime})^2 = 0$$
(15)

$$\phi^{\prime\prime} + \frac{Le.f}{2} \frac{\phi^{\prime}}{2} + \frac{N_t}{N_b} \theta^{\prime\prime} = 0 \tag{16}$$

The transformed boundary conditions are given by  $Pr.f(0) + 2 M \theta'(0) = 0, \quad f'(0) = \gamma, \quad \theta(0) = 1, \quad \phi(0) = 1$   $f'(\infty) = 1 - \gamma, \quad \theta(\infty) = 0, \qquad \emptyset(\infty) = 0$  (17) where prime denote differentiation with respect to  $\eta$  and the four parameters are defined by

$$Pr = \frac{v}{\alpha}, \qquad Le = \frac{v}{D_B}, M = \frac{C_P (T_\infty - T_m)}{\lambda + C_S (T_m - T_0)}$$

$$N_b = \frac{(\rho c)_P D_B (\phi_W - \phi_\infty)}{(\rho c)_f v},$$

$$N_t = \frac{(\rho c)_P D_T (T_m - T_\infty)}{(\rho c)_f T_\infty v}$$
(18)

Here, *Pr, Le,*  $N_b$  and  $N_t$  denote the Prandtl number, the Lewis number, the Brownian motion parameter and the thermophoresis parameter respectively. It is important to note that this boundary value problem reduces to the classical problem of flow and heat and mass transfer due to a stretching surface in a viscous fluid when N<sub>b</sub> and N<sub>t</sub> are zero.

The quantities of practical interest, in this study, are the Nusselt number Nu and the Sherwood number Sh which are defined as:

Friction Factor

$$C_{\infty} = \frac{\tau_{w}}{\left(\frac{\rho u_{\infty}^{2}}{2}\right)} = 2 R e_{\infty}^{-\frac{1}{2}} (1 - \gamma)^{-\frac{3}{4}} |f''(0)|$$

$$C_{w} = \frac{\tau_{w}}{\left(\frac{\rho u_{w}^{2}}{2}\right)} = 2 R e_{w}^{-\frac{1}{2}} \gamma^{-\frac{3}{4}} |f''(0)|$$
(19)

The local heat transfer rate (Local Nusselt number) is given by

$$Nu_{x} = \frac{q_{w}x}{R(T_{w} - T_{\infty})} = -\frac{R\theta'(0)}{R} \frac{(u_{w} - u_{\infty})^{\frac{1}{2}}}{\theta^{-\frac{1}{2}x^{-\frac{1}{2}}}} x = -(Re_{w} - Re_{\infty})^{\frac{1}{2}} \theta'(0)$$
(20)

Similarly the local Sherwood number is given by  

$$Sh_x = \frac{q_m x}{P_n(c_m - Re_w)^2} = -(Re_w - Re_w)^{\frac{1}{2}} \phi'(0)$$
(21)

$$Sh_x = \frac{1}{D_B(C_W - c_\infty)} = -(Re_W - Re_\infty)^2 \varphi(0)$$
(2)

where  $q_w$  and  $q_m$  are wall heat and mass flux rates, respectively.

### 3. Results and discussions

The nonlinear ordinary differential equations (14)-(16), satisfying the boundary conditions (17) were integrated numerically by using the fourth-order Runge-Kutta scheme along with the shooting method for several values of the governing parameters, namely, Prandtl number (Pr), Lewis number (Le), Brownian motion parameter (N<sub>b</sub>) and thermophoresis parameter (N<sub>t</sub>). It may be noted that Pr = 0.007 corresponds to liquid metal Gallium and Pr = 0.025 to liquid metal germanium. In order to assess the accuracy of the present results, we obtained results for the reduced Nusselt number  $-\theta'(0)$  by ignoring the effects of M, N<sub>b</sub> and N<sub>t</sub>. These results are shown in Table 1. A comparison of our results with literature values indicates excellent agreement and therefore our results are highly accurate.

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Pr	Present results	Wang [14]	Gorla and Sidawi [15]
0.07	0.0656	0.0656	0.0656
0.20	0.1691	0.1691	0.1691
0.70	0.4539	0.4539	0.4539
7.00	1.8907	1.8954	1.8905
20.00	3.3539	3.3539	3.3539
70.00	6.4622	6.4622	6.4622

Table 1: Comparison of results for  $-\theta'(0)$  for the Sakiadis problem (M = N<sub>t</sub> = N<sub>b</sub> = 0;  $\gamma = 1$ )

Table 2: Effects of M on f"(0),  $-\dot{\theta}(0)$  and  $-\phi(0)$  for N<sub>b</sub>=0.3, N<sub>t</sub>=0.1, Le=10, Pr=0.007 and  $\gamma$ =0.1

М	f"(0)	-θ <sup>`</sup> (0)	-φ <sup>'</sup> (0)
0	2.783765E-01	4.241628E-02	8.482305E-01
0.5	3.170545E-01	4.246388E-02	1.310642
1.0	3.547819E-01	4.313251E-02	1.811196
2.0	4.280522E-01	4.374699E-02	2.846745
3.0	4.985808E-01	4.482413E-02	3.868566

Tables 2 displays the resulting values of velocity gradient f''(0), the sheet surface heat transfer rate  $-\phi'(0)$ and the mass transfer rate  $-\phi'(0)$ , which are proportional to the friction factor, Nusselt number and Sherwood number respectively, for several values of the melting parameter, M. The results indicate that as M increases, the friction factor, heat transfer rate and mass transfer rate increase. Thus, increasing the melting strength increases the heat and mass transfer rates at the solid/fluid interface. Melting provides a blowing condition at the moving surface. The results in Table 3 show that as the velocity ratio  $\gamma$  increases, the heat transfer rates decrease and mass transfer rates increase. The results in Table 4 show that as the Lewis number, Le increases, the heat transfer and mass transfer rates decrease and mass transfer rates increase. The results in Table 5 show that as the thermophoresis parameter Nt increases, the heat transfer rates decrease and mass transfer rates decrease and mass transfer rates decrease and mass transfer rates increases the heat transfer rates decrease and mass transfer rates increases the heat transfer rates increases the heat transfer rates increases the heat transfer rates increases and mass transfer rates increases and mass transfer rates decrease and mass transfer rates increases the heat transfer rates increases the heat transfer rates increases the heat transfer rates increases and mass transfer rates increases the heat transfer rates increases and mass transfer rates increases the heat transfer rates decrease and mass transfer rates increases increases increases. The results in Table 6 show that as the Brownian motion parameter (N<sub>b</sub>) increases, the heat transfer rates decrease and mass transfer rates increases. As the Prandtl number Pr increases, the friction factor increases and in consequence increases the heat transfer rate at the surface.

Table 3: Effects of  $\gamma$  on f"(0),  $-\dot{\theta}(0)$  and  $-\dot{\phi}(0)$  for M=1.0, N<sub>b</sub>=0.3, N<sub>t</sub>=0.1, Le=10 and Pr=0.007

γ	f''(0)	-θ <sup>`</sup> (0)	-φ <sup>`</sup> (0)
0	4.020455E-01	4.457481E-02	1.393198
0.25	2.616510E-01	4.082558E-02	2.674506
0.5	-8.41530E-05	3.644821E-02	5.225882
0.75	-5.92690E-01	3.191911E-02	10.236470
1.0	-1.987978	2.772456E-02	18.883270

Table 4: Effects of L on f''(0),  $-\theta'(0)$  and  $-\phi'(0)$  for M=1.0, N<sub>b</sub>=0.3, N<sub>t</sub>=0.1, Pr=0.007 and  $\gamma$ =0.1

Le	f''(0)	-θ <sup>'</sup> (0)	-φ <sup>'</sup> (0)
1	3.547975E-01	4.321312E-02	4.426402E-01
10	3.547819E-01	4.313251E-02	1.811196
$10^{2}$	3.547536E-01	4.342895E-02	13.631110
$10^{3}$	3.547498E-01	4.330121E-02	132.718400

N <sub>t</sub>	f''(0)	-θ <sup>'</sup> (0)	-φ <sup>(0)</sup>
0.1	1.501518E-01	2.820203E-01	1.063284
0.2	1.501518E-01	2.690583E-01	1.063858
0.3	1.501518E-01	2.571833E-01	1.068012
0.4	1.501518E-01	2.457135E-01	1.075496
0.5	1.501518E-01	2.348450E-01	1.086077

Table 5: Effects of N<sub>t</sub> on f"(0),  $-\theta'(0)$  and  $-\phi'(0)$  for M=0.5, N<sub>b</sub>=0.3, Le=10, Pr=1.0 and  $\gamma$ =0.1

Table 6: Effects of N<sub>b</sub> on f"(0),  $-\dot{\theta}(0)$  and  $-\dot{\phi}(0)$  for M=0.5, N<sub>t</sub>=0.3, Le=10, Pr=1.0 and  $\gamma$ =0.1

N <sub>b</sub>	f''(0)	-θ <sup>`</sup> (0)	-φ <sup>'</sup> (0)
0.1	1.501518E-01	3.021600E-01	9.746230E-01
0.2	1.501518E-01	2.791630E-01	1.045009
0.3	1.501518E-01	2.571833E-01	1.068012
0.4	1.501518E-01	2.369526E-01	1.078316
0.5	1.501518E-01	2.185187E-01	1.083437

Figures 2-4 display results for the variation of velocity, temperature and concentration within the boundary layer. As the melting parameter M increases, the velocity increases whereas the temperature and concentration decrease. Figure 5 shows the velocity distribution within the boundary layer for several values of the velocity ratio parameter  $\gamma$ . A value of zero for the velocity parameter  $\gamma$  describes the Blasius problem whereas  $\gamma = 1$  describes the Sakiadis problem. Figure 6 and 7 show the temperature and concentration distribution within the boundary layer for several values of the velocity ratio parameter  $\gamma$ . As  $\gamma$  increases, the temperature increases while the concentration values decrease. Figures 8 and 9 show the temperature and concentration distribution as the Lewis number increases. As Le increases, we observe that the concentration decreases and the concentration boundary layer thickness decreases. This in turn increases the surface mass transfer rates as Le increases.



Figure 2: Effects of M on velocity profiles



Figure 3: Effects of M on temperature profiles



1.00 M=1.0 N\_=0.3 γ=0 N=0.1 0.75 γ=0.25 Le=10 Pr=0.007 γ=0.5 0.50 £, γ=0.75 0.25 γ=1.0 0.00 2 4 η

Figure 4: Effects of M on solid volume fraction profiles



Figure 6: Effects of  $\gamma$  on temperature profiles



Figure 8: Effects of Le on temperature profiles





Figure 7: Effects of  $\gamma$  on solid volume fraction profiles



Figure 9: Effects of Le on solid volume fraction profiles

1.0

Figures 10 and 11 show the variation of temperature and concentration as  $N_t$  was chosen as prescribable parameter. As  $N_t$  increases, the temperature and concentration increase.





M=0.5

Figure 10: Effects of Nt on temperature profiles

Figure 11: Effects of N<sub>t</sub> on solid volume fraction profiles

Figures 12 and 13 show the variation of temperature and concentration as  $N_b$  was chosen as prescribable parameter. As  $N_b$  increases, the temperature increases whereas the concentration decreases.



Figure 12. Effects of N<sub>b</sub> on temperature profiles



Figure 13: Effects of  $N_b$  on solid volume fraction profiles

#### 4. Concluding Remarks

In this work, we have studied the problem of the steady boundary layer flow of a warm, laminar nanoliquid over a melting surface moving parallel to a uniform free stream of the same material. The governing boundary layer equations are solved numerically using the fourth-order Runge-Kutta scheme along with the shooting method. The development of the Nusselt number and Sherwood number as well as the temperature, concentration and velocity distributions for various values of the velocity ratio, melting and nanofluid parameters has been discussed and illustrated in tabular forms and graphs. The results indicate that the melting phenomenon increases the heat transfer rate, (Nusselt number) at the solid/fluid interface. The effect of the

nanofluid parameters on the temperature and concentration distributions as well as the friction factor and heat and mass transfer depends on the ratio of the velocity of the plate and the free stream fluid velocity.

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